

# Session 22

2032  
1129-1

Linearized eq of motion for holonomic systems

$$\underline{x} = \underline{q} - \underline{q}_0, \quad \underline{M} \ddot{\underline{x}} + (\underline{G} + \underline{C}) \dot{\underline{x}} + (\underline{K} + \underline{B}) \underline{x} = \underline{0}$$

$$-\frac{\partial V}{\partial \underline{q}} \Big|_{\underline{q}_0} = \underline{Q} \Big|_{\underline{q}_0}$$

In the conservative & natural case  
(potential)

$$\underline{M} \ddot{\underline{x}} + \underline{K} \underline{x} = \underline{0} \quad (1)$$

Simplest derivation from quadratic Lagrangian

$$L = \frac{1}{2} \dot{\underline{x}}^T \underline{M} \dot{\underline{x}} - \frac{1}{2} \underline{x}^T \underline{K} \underline{x} + O(\epsilon)$$

Example



Spring is unstretched at  $\phi_1 = \phi_2 = 0$

Assume. Unstretched length = 0

$$T = \frac{1}{2} m l^2 (\dot{\phi}_1^2 + \dot{\phi}_2^2); \quad \text{Equilibrium. } \phi_{10} = \phi_{20} = 0 \Rightarrow \underline{x} = \underline{\phi} - \underline{\phi}_0 = \underline{\phi}$$

$$- \frac{1}{2} m l^2 (\dot{x}_1^2 + \dot{x}_2^2)$$

$$V = mgl(1 - \cos \phi_1) + mgl(1 - \cos \phi_2) + \frac{1}{2}k \left[ (h \cos \phi_1 - h \cos \phi_2)^2 + (h \sin \phi_1 - h \sin \phi_2)^2 \right]$$

$$\Rightarrow L = \frac{1}{2}ml^2(\dot{x}_1^2 + \dot{x}_2^2) - mgl \left[ 1 - \left(1 - \frac{\phi_1^2}{2} + \dots\right) + 1 - \left(1 - \frac{\phi_2^2}{2} + \dots\right) \right] - \frac{1}{2}kh^2 \left[ \left(1 - \frac{\phi_1^2}{2} + \dots - 1 + \frac{\phi_2^2}{2} + \dots\right)^2 + (\phi_1 - \phi_2 + \dots)^2 \right]$$

$$= \frac{1}{2}ml^2(\dot{x}_1^2 + \dot{x}_2^2) - \frac{1}{2}mgl(\phi_1^2 + \phi_2^2) - \frac{1}{2}kh^2(x_1 - x_2)^2 + O(4)$$

Sym, pos. def.

$$\Rightarrow M = \begin{pmatrix} ml^2 & 0 \\ 0 & ml^2 \end{pmatrix}; \quad K = \begin{pmatrix} mgl + kh^2 & -kh^2 \\ -kh^2 & mgl + kh^2 \end{pmatrix} \rightarrow \text{Stiffness}$$

(mass matrix)

if we expand about another equilibrium point, we'll get different matrices

Since the equilibrium is stable  $\rightarrow K$  has to be positive definite and it is

Systematic Solution of (1):

Trial Solution  $\tilde{x}(t) = \underline{x}_0 e^{\lambda t}$ ; general Solution  $x(t) = \sum c_i \tilde{x}_i(t)$

$\underline{x}_0 \neq 0$

From initial Conditions

plug into (1):

$$(2) \quad (\lambda^2 \underline{M} + \underline{K}) \underline{x}_0 = 0, \text{ let } \underline{x}_0 = \underline{a}$$

Note: if  $\underline{a}$  is a solution, so is  $c\underline{a} \Rightarrow$  mode shapes are never unique in magnitude

left multiply (2) by  $\underline{a}^T \Rightarrow \lambda^2 = - \frac{\underline{a}^T \underline{K} \underline{a}}{\underline{a}^T \underline{M} \underline{a}}$

Case (a):  $\underline{K} = \frac{\delta^2 V}{\delta \underline{q}^2} \Big|_{\underline{q}_0}$  is positive definite

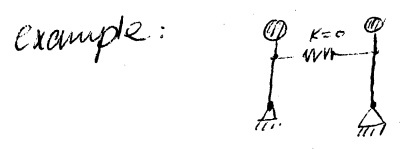
$$\Rightarrow \lambda^2 < 0 \Rightarrow \lambda = \pm i\omega \Rightarrow \begin{cases} \tilde{x}_1(t) = \underline{a} e^{i\omega t} \\ \tilde{x}_2(t) = \underline{a} e^{-i\omega t} \end{cases}$$

$\tilde{x}(t)$ : normal mode  
 $\underline{a}$ : mode shape  
 $\omega$ : natural frequency

Case (b)  $\underline{K} = \frac{\partial^2 V}{\partial q^2} \Big|_{q_0}$  is negative definite  $\Rightarrow$  maximum for  $V$  at  $q_0$

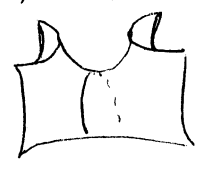
$\Rightarrow \lambda^2 > 0 \Rightarrow \lambda = \pm \nu \Rightarrow \begin{cases} \tilde{x}_1(t) = a e^{\nu t} \\ \tilde{x}_2(t) = a e^{-\nu t} \end{cases}$

$\Rightarrow \alpha = 0$  is unstable



Case (c):  $\underline{K} = \frac{\partial^2 V}{\partial q^2} \Big|_{q_0}$  is indefinite  $\rightarrow V$  has a saddle type nature (has a point of inflection)

Instability



Example: Coupled pendulum & spring  $K \neq 0$

How do we find  $\lambda$  &  $\alpha$ ?

Consider the case of pos. def.  $\underline{K} \Rightarrow (-\omega^2 \underline{M} + \underline{K}) \underline{a} = \underline{0}$   
 $\Rightarrow -\omega^2 \underline{M} + \underline{K}$  is a

Singular matrix  $\Rightarrow \begin{vmatrix} -\omega^2 m_{11} + K_{11} & & -\omega^2 m_{1n} + K_{1n} \\ \vdots & \ddots & \vdots \\ -\omega^2 m_{n1} + K_{n1} & & -\omega^2 m_{nn} + K_{nn} \end{vmatrix} = 0$

$\cdot n^{\text{th}}$  order polynomial for  $\omega^2$  (characteristic equation)

$a_1 (\omega^2)^n + a_2 (\omega^2)^{n-1} + \dots + a_{n+1} = 0$

always has  $n$ -roots (all roots are real here - since  $\underline{M}, \underline{K}$  are pos. def.)

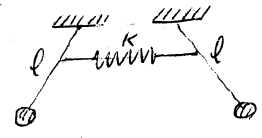
$\Rightarrow \omega_1^2 \dots \omega_n^2$   
 $\Rightarrow \omega_1, \dots, \omega_n$

$n$ -natural frequencies

Finding the mode shapes:  $(-\omega^2 \underline{M} + \underline{K}) \underline{a} = \underline{0} \Rightarrow \underline{a}_j = \dots$

terminology  $\Phi = [\underline{a}_1, \dots, \underline{a}_n]$  modal matrix (is not unique, but its directions are unique)

Example: Re Consider pendulum-Spring system



$$\underline{M} = \begin{pmatrix} ml^2 & 0 \\ 0 & ml^2 \end{pmatrix} \quad \underline{K} = \begin{pmatrix} mgl + Kh^2 & -Kh^2 \\ -Kh^2 & mgl + Kh^2 \end{pmatrix}$$

Let:  $ml^2 = 1$  [kg·m<sup>2</sup>]  
 $mgl = 2$  [N·m]  
 $Kh^2 = 1$  [N·m]

Char. eq.  $\begin{pmatrix} -\omega^2 + 3 & -1 \\ -1 & -\omega^2 + 3 \end{pmatrix} = \omega^4 - 6\omega^2 + 8 = 0$   
 $\Rightarrow \omega_1 = \sqrt{2}$  [ $\frac{1}{s}$ ]  
 $\omega_2 = 2$  [ $\frac{1}{s}$ ]

Mode shapes: (1)  $\omega_1^2 = 2$

$$\begin{pmatrix} -2+3 & -1 \\ -1 & -2+3 \end{pmatrix} \begin{pmatrix} a_1 \\ a_2 \end{pmatrix} = \underline{0}$$

$$\vec{a}_1 = \begin{pmatrix} 1 \\ 1 \end{pmatrix}$$

(2)  $\omega_2^2 = 4$

$$\begin{pmatrix} -4+3 & -1 \\ -1 & -4+3 \end{pmatrix} \begin{pmatrix} a_1 \\ a_2 \end{pmatrix} = \underline{0} \rightarrow \vec{a}_2 = \begin{pmatrix} 1 \\ -1 \end{pmatrix}$$

